

# Practice-Relevant Teaching of Gas Turbine Performance

GPPS-TC-2025-0109 Presentation by Joachim Kurzke



# Commercial Airliner 1958

## JT3C Turbojet



Should teaching gas turbine performance begin with the turbojet?

Turbojets are in 2025 not practice-relevant for commercial airliners

That's what I said during the presentation



# Commercial Airliner 2023

## CFM Leap Turbofan



Turbofans are  
practice-relevant  
now!

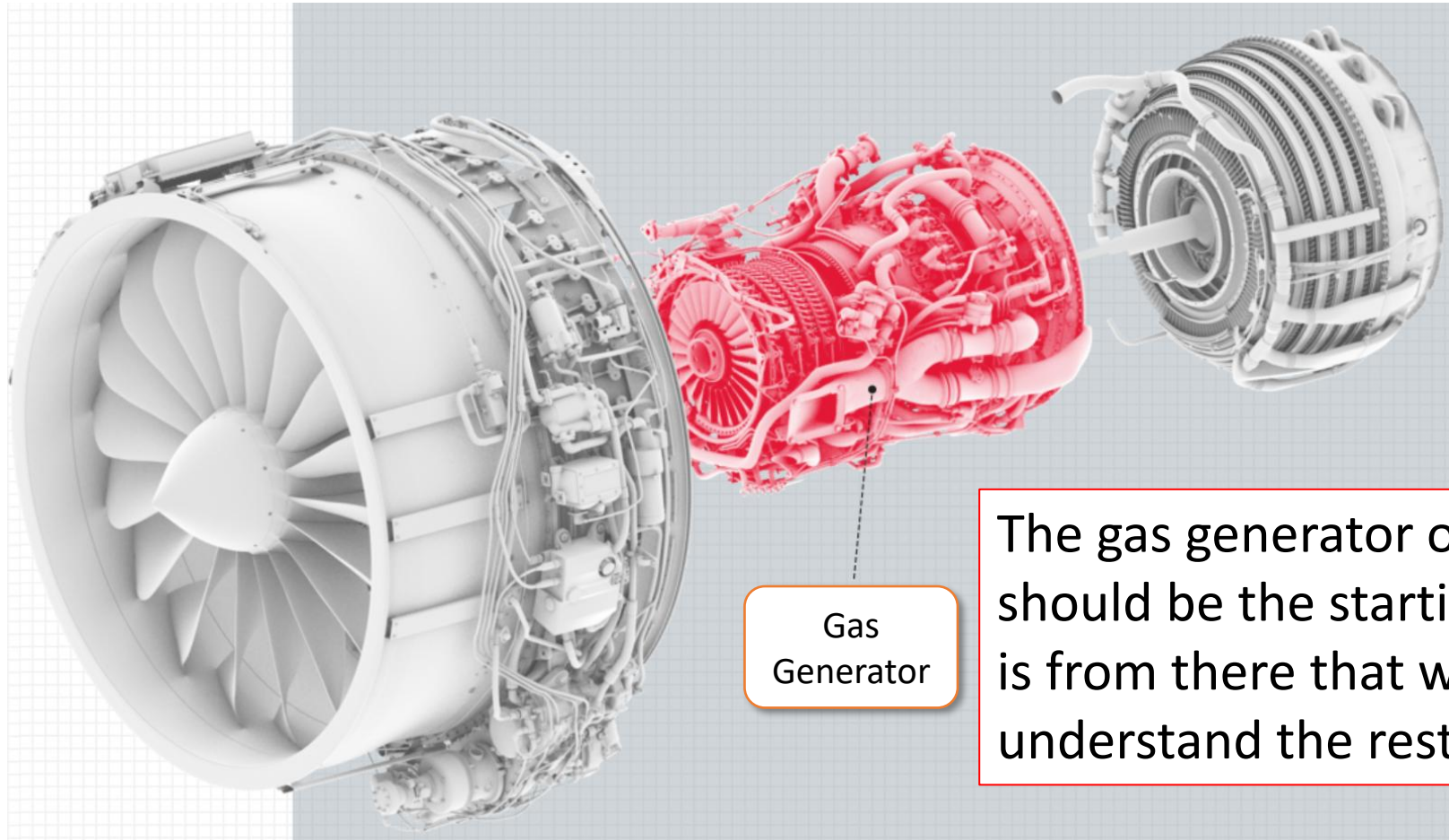
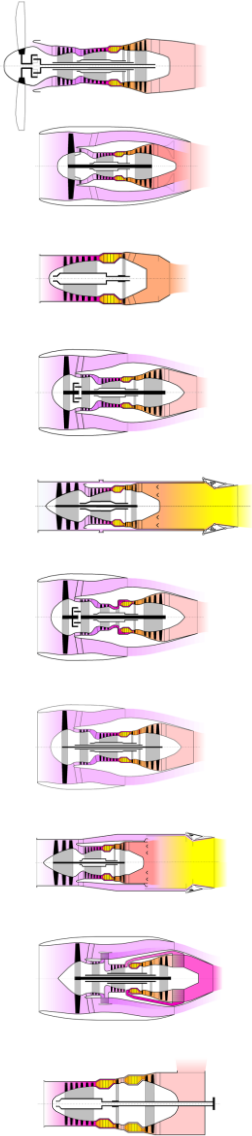
Could teaching  
also begin with  
the turbofan?

Yes!





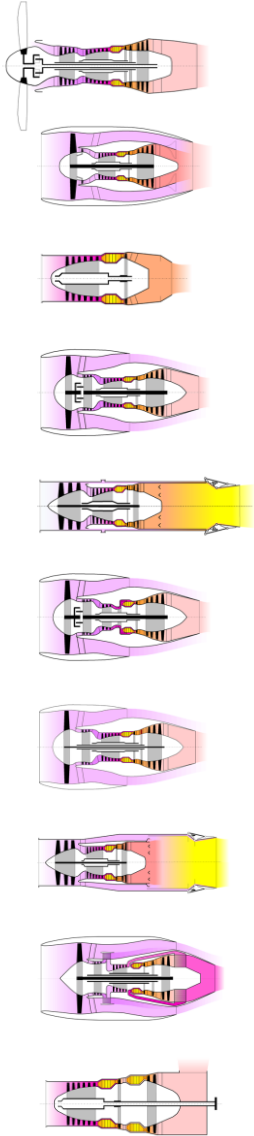
# CFM56-3 Modular Design



The heart of any aircraft gas turbine configuration is a gas generator!

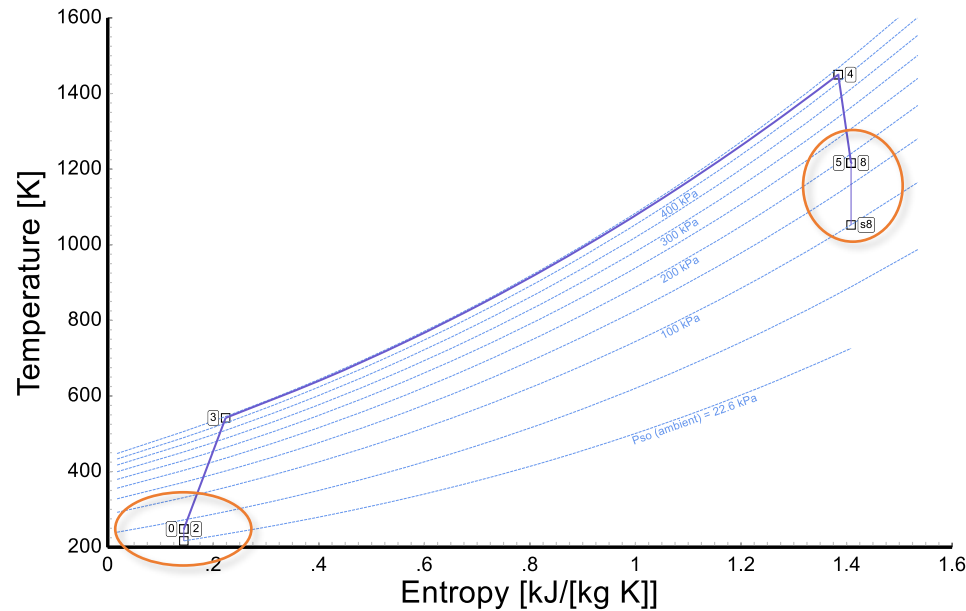
The gas generator of the turbofan should be the starting point, and it is from there that we can begin to understand the rest of the system!





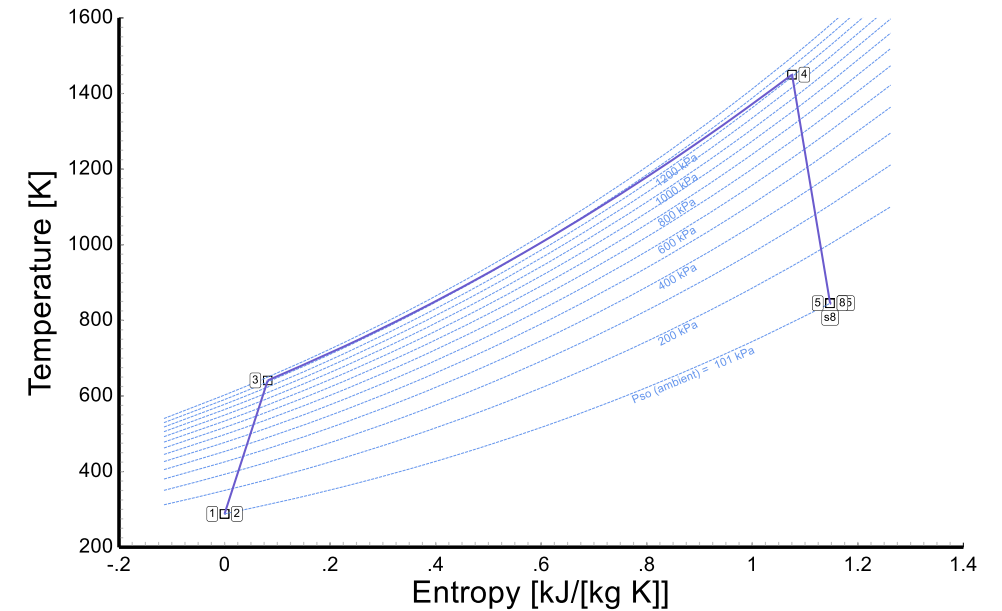
# Introduction to the Thermodynamics of Gas Turbines

Turbojet @ 36000ft/0.85



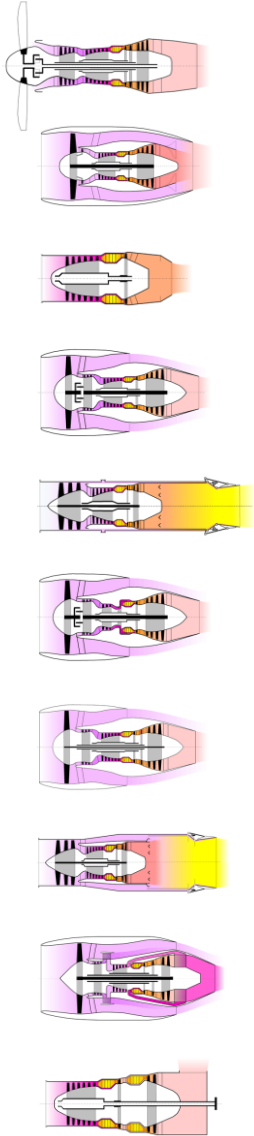
In fact, the operating cycle of a turbojet at 36,000 feet and Mach 0.85 is more complex than that of a gas generator!

Turboshaft SLS



The simplest way to introduce students to the field of gas turbine performance is to model the cycle of the gas generator as a turboshaft on a sea-level test bed!

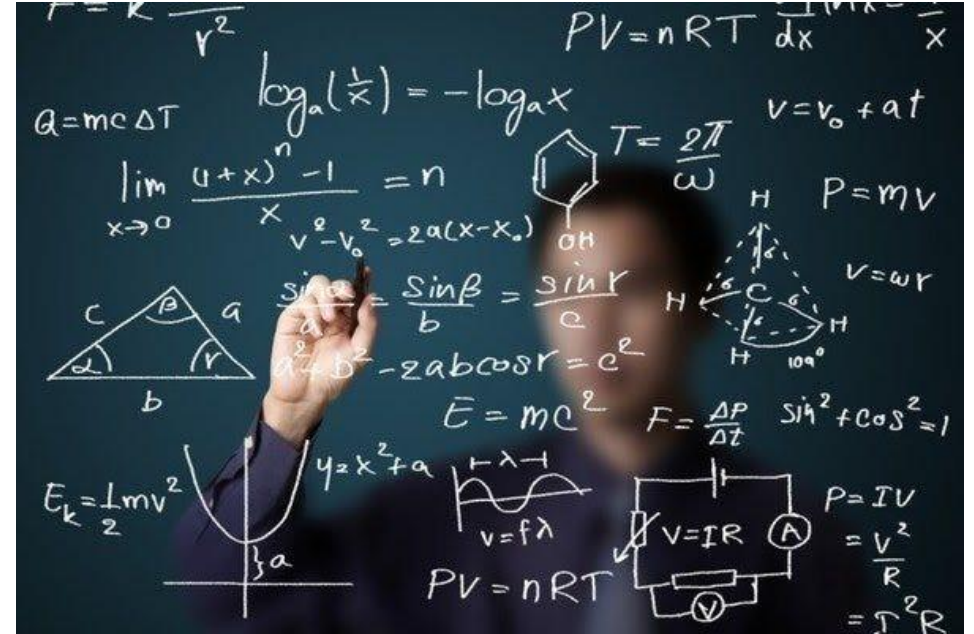




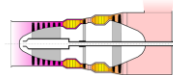
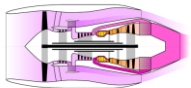
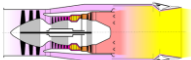
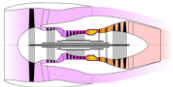
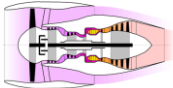
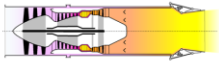
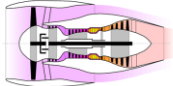
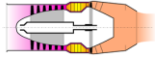
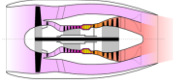
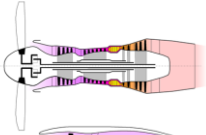
# About Equations

... are they suited for realistic gas turbine performance calculations?

No one uses equations  
in practice!







# Thermal Efficiency

## The Academic Definition

$$\eta_{th} = \frac{\Delta H_T - \Delta H_C}{\Delta H_B}$$

$\Delta H$  = specific work in  
kW per kg/s mass flow

$$\eta_{th} = \frac{\frac{\gamma_T}{\gamma_T - 1} * R_T * \frac{T_4}{T_2} * \left[ 1 - \left( \frac{P_2}{P_3} \right)^{\frac{\gamma_T - 1}{\gamma_T}} \right] * \eta_T - \frac{\gamma_C}{\gamma_C - 1} * R_C * \left[ \left( \frac{P_3}{P_2} \right)^{\frac{\gamma_C - 1}{\gamma_C}} - 1 \right] / \eta_C}{\left( \frac{T_4}{T_2} - \frac{T_3}{T_2} \right) * \left( \frac{\gamma_C}{\gamma_C - 1} * R_C + \frac{\gamma_T}{\gamma_T - 1} * R_T \right) / 2}$$

This equation uses different isentropic exponents  $\gamma_C, \gamma_T$  and gas constants  $R_C, R_T$  for the compressor and turbine. Efficiencies  $\eta_C, \eta_T$  are isentropic.



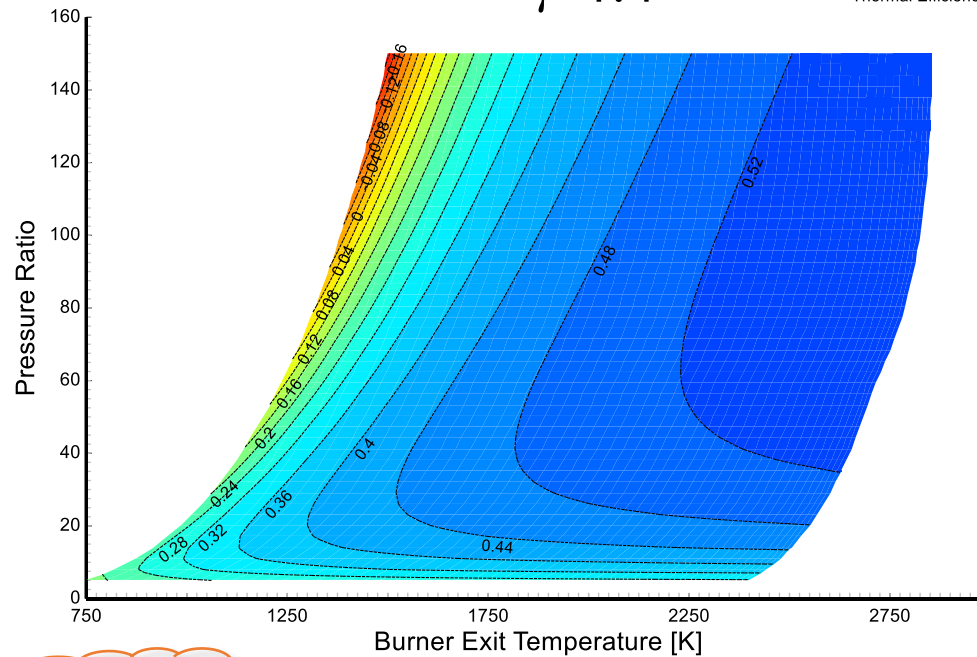
# Thermal Efficiency – Academic Definition

## Effect of Isentropic Exponent

With constant isentropic exponent:  
The higher the burner exit temperature, the greater the thermal efficiency.

$$\gamma = 1.4$$

Thermal Efficiency

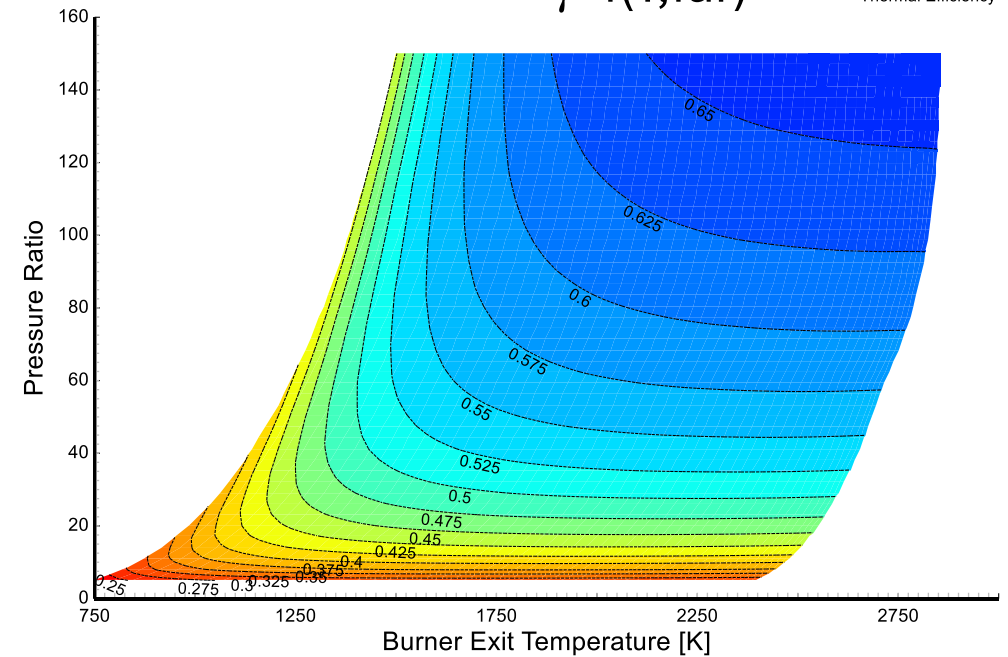


Color represents  
thermal efficiency

With true gas properties:  
The higher the pressure ratio, the greater the thermal efficiency.

$$\gamma = f(T, \text{far})$$

Thermal Efficiency



$$\eta_{C,\text{pol}} = 0.9 \quad \eta_{T,\text{pol}} = 0.9$$



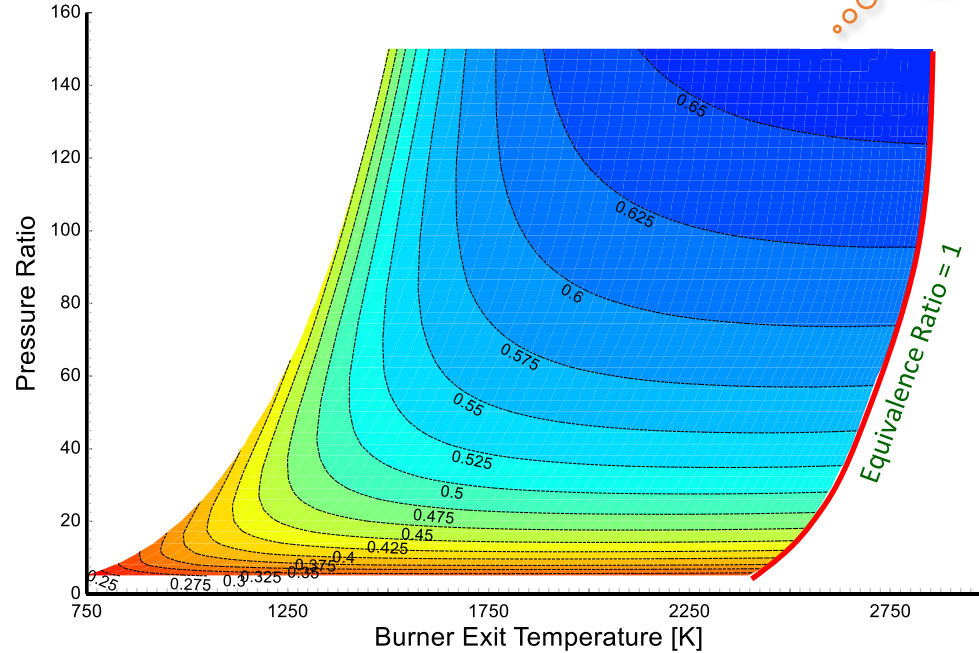


# Two Definitions of Thermal Efficiency

## Academic

$$\eta_{th} = \frac{\Delta H_T - \Delta H_C}{\Delta H_B}$$

No optimum exists!



$$\gamma = f(T, \text{far})$$

$$\eta_{C, \text{pol}} = 0.9$$

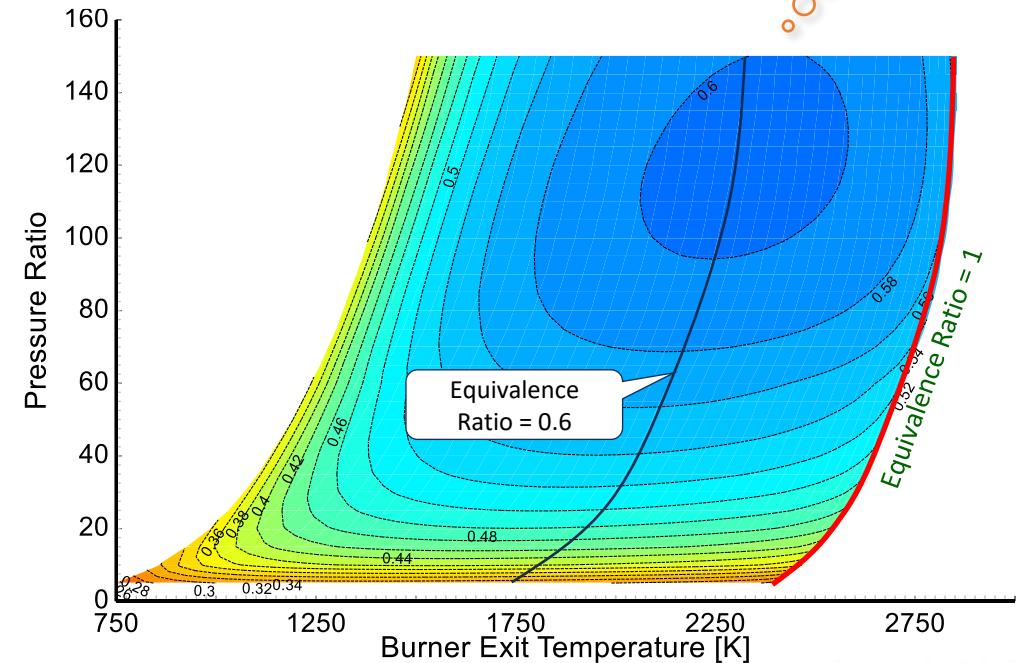
$$\eta_{T, \text{pol}} = 0.9$$

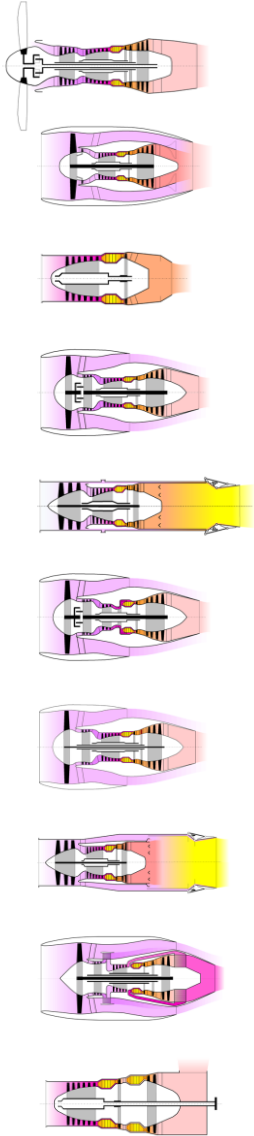
$$\text{Equivalence ratio} = \frac{\text{far}}{\text{far}_{\text{stoichiometric}}}$$

## Practice

$$\eta_{th} = \frac{\Delta H_T - \Delta H_C}{W_F \times FHV}$$

There is an optimum along the line where the equivalence ratio is equal to 0.6.





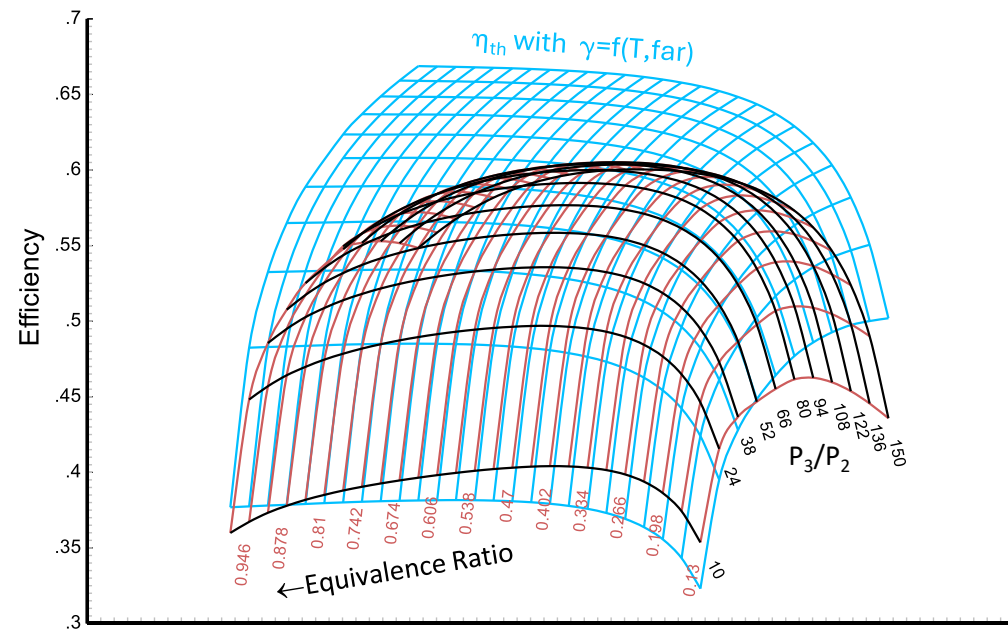
# Thermal Efficiency – The Numbers

Academic

$$\eta_{th} = \frac{\Delta H_T - \Delta H_C}{\Delta H_B}$$

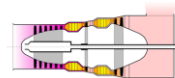
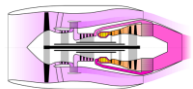
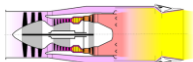
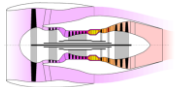
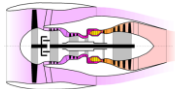
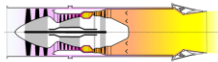
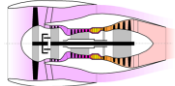
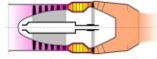
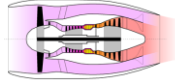
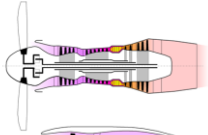
Practice

$$\eta_{th} = \frac{\Delta H_T - \Delta H_C}{W_F \times FHV}$$



The academic definition of thermal efficiency yields numbers that are not relevant in practice.



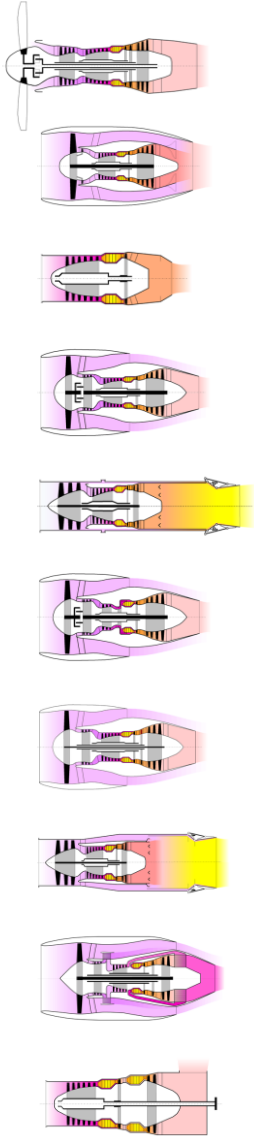


# Observations

- The result for constant  $\gamma$  is misleading because it says that the highest burner exit temperature gives the highest thermal efficiency.
- The result for  $\gamma=f(T, \text{far})$  is misleading also because it says that the highest pressure ratio gives the highest thermal efficiency.
- The pressure ratios of up to 150 considered have nothing in common with those of real gas turbines.
- The heat addition method is not included in the academic definition of thermal efficiency.
  - In practice, heat is added by burning fuel.
  - The achievable temperature is limited by stoichiometric combustion
  - The slope  $\Delta T / \Delta W_f$  decreases with increasing fuel-air-ratio until  $\Delta T / \Delta W_f = 0$  for stoichiometric combustion

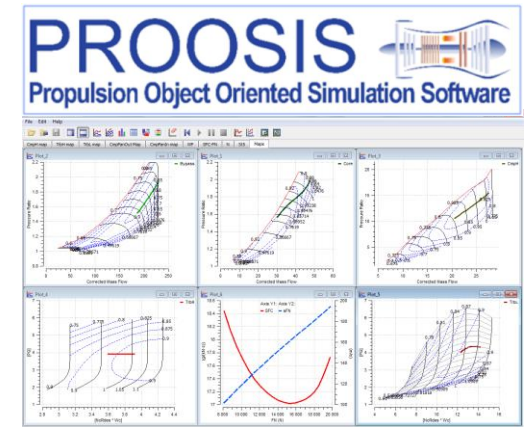








# Performance Software

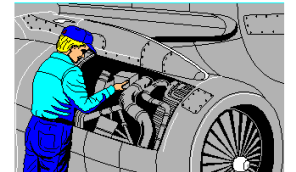
The tool used in practice



## GasTurb 8 Professional

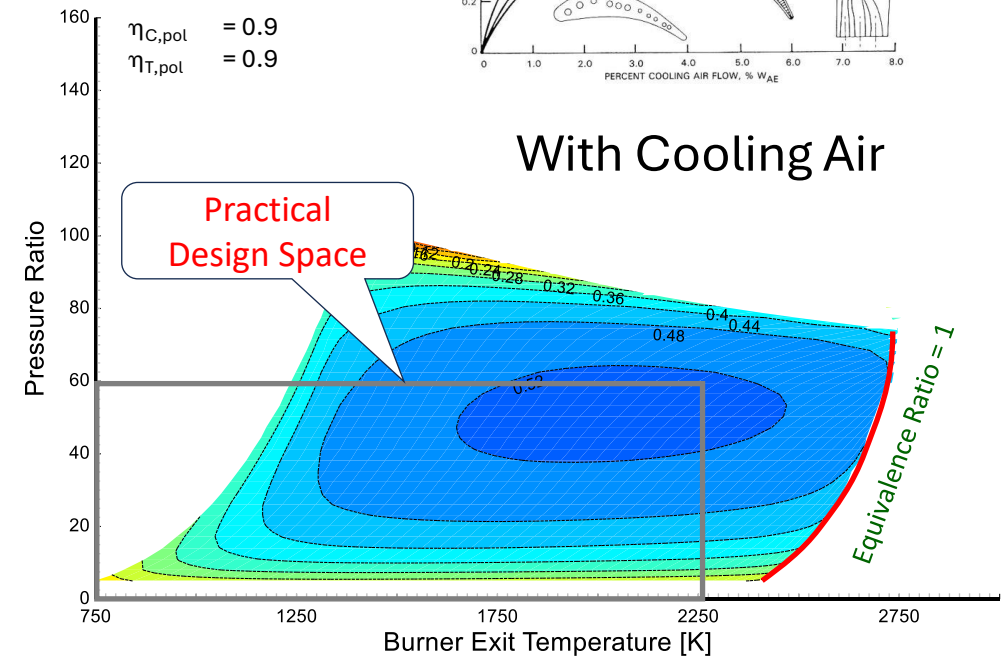
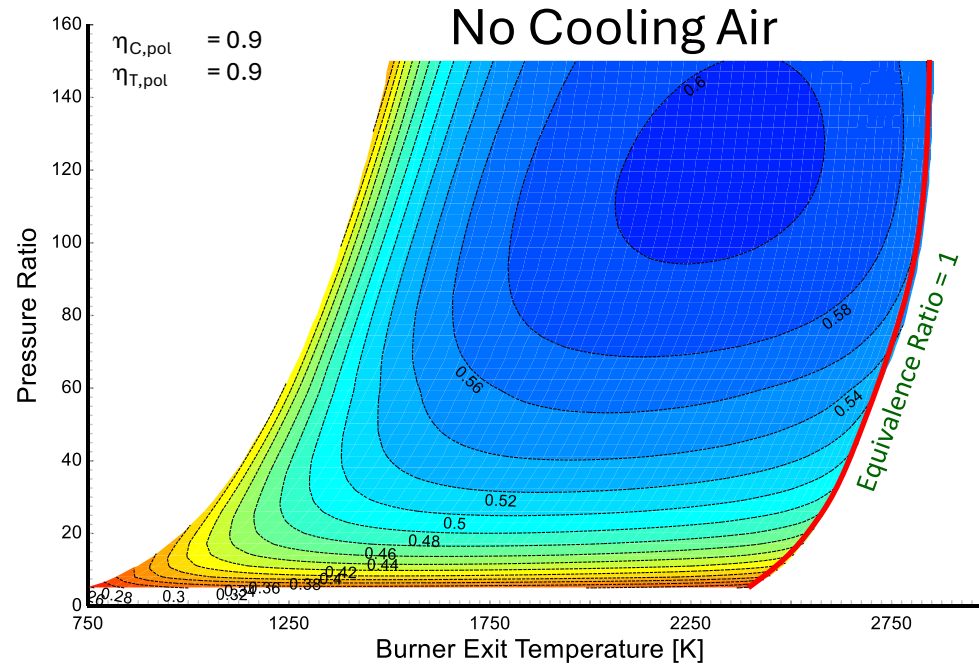
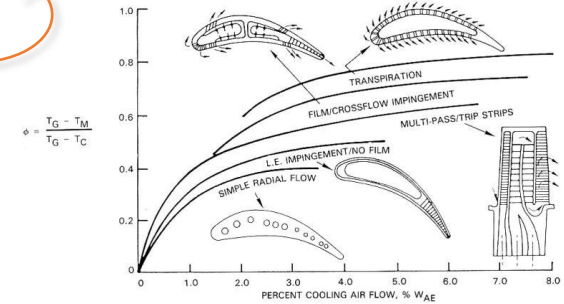
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-  [GasTurb 8](#)
-  [GasTurb Details](#)
-  [Smooth C Demo](#)



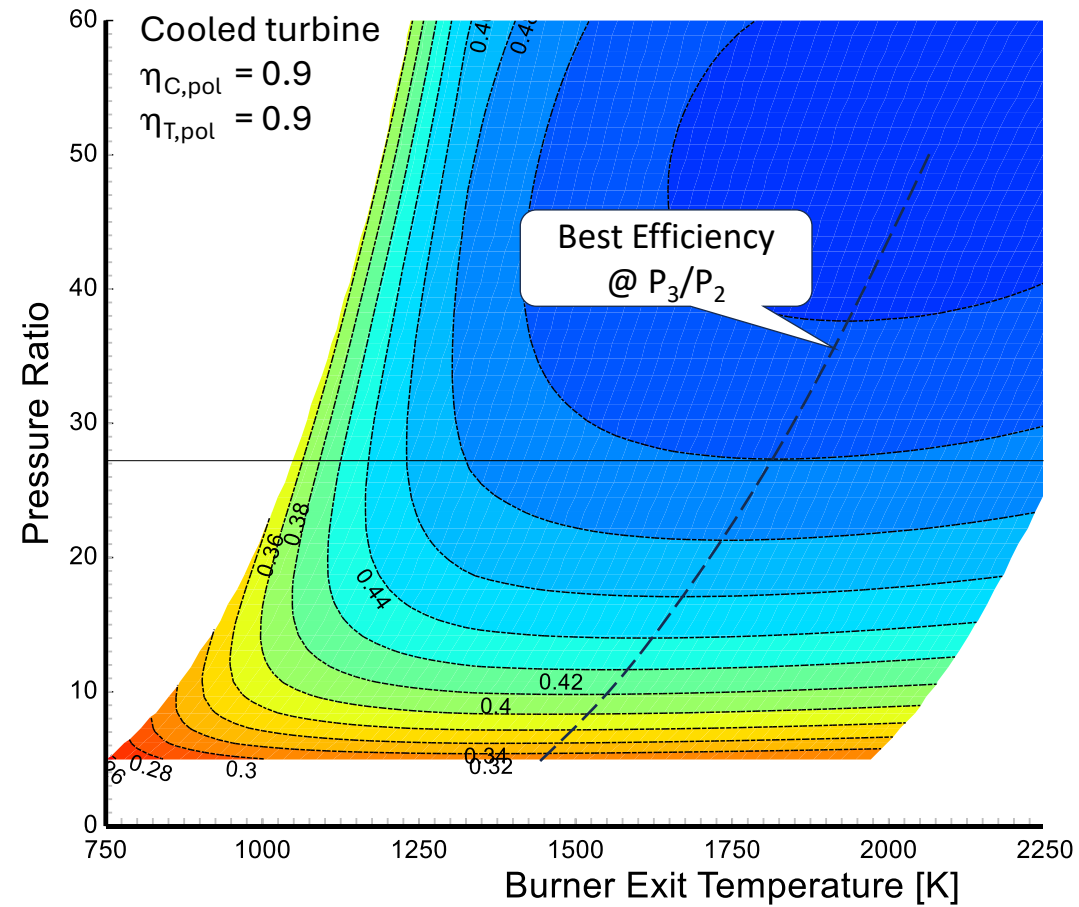
# A More Realistic Efficiency Calculation

The amount of cooling air needed depends on both the gas temperature and the cooling air temperature.



# Practical Design Space

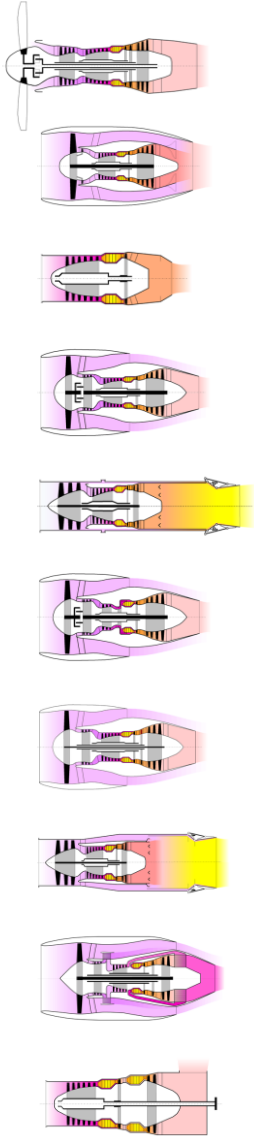
## Thermal Efficiency



Stoichiometric combustion  
does not yield the ultimate  
thermal efficiency!

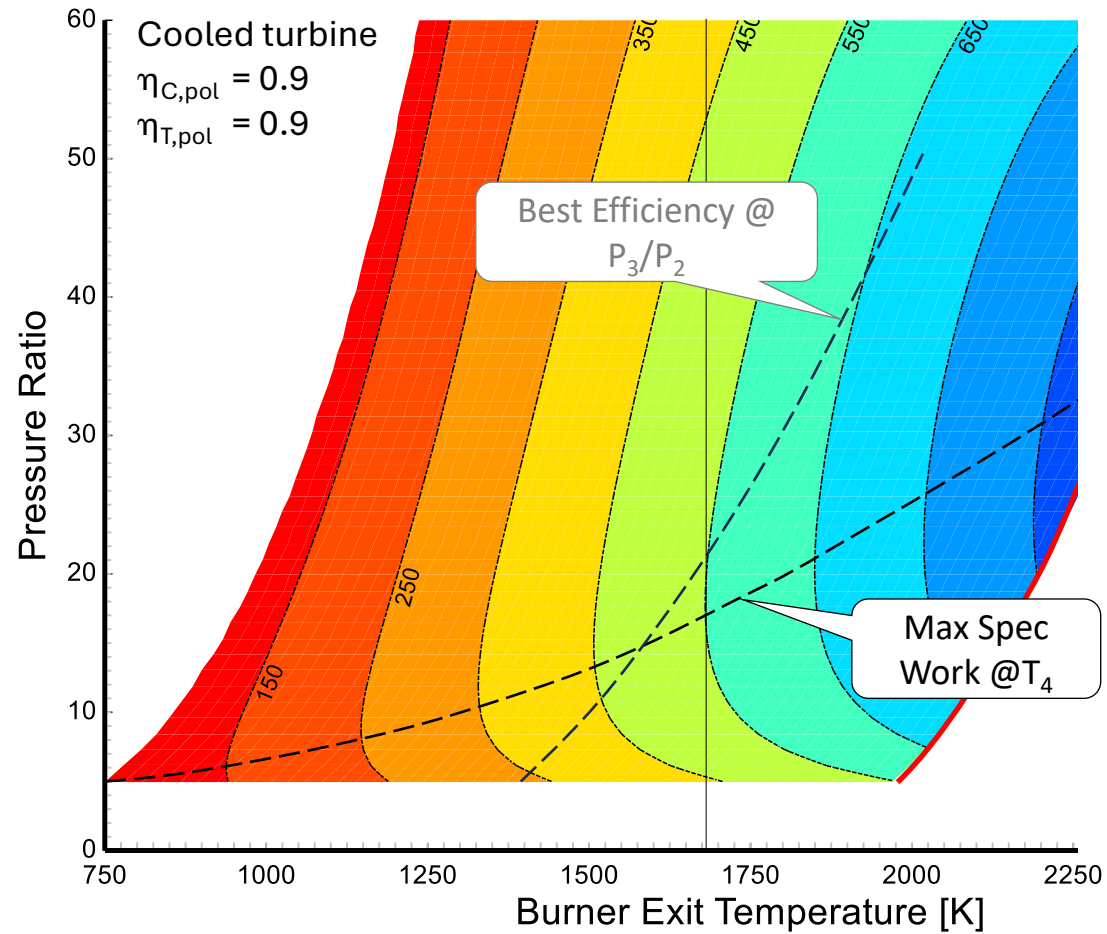


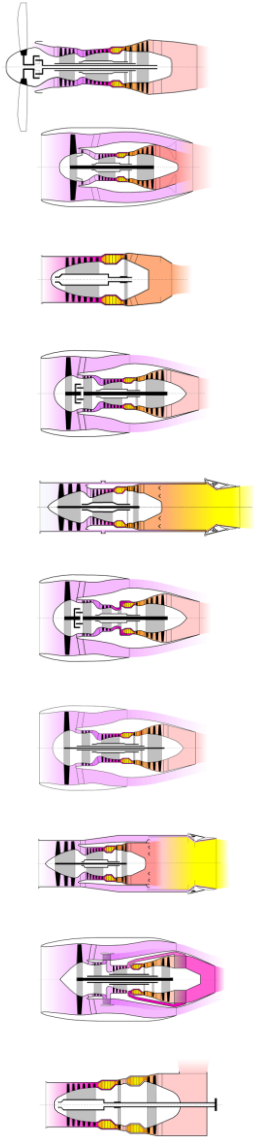




# Practical Design Space

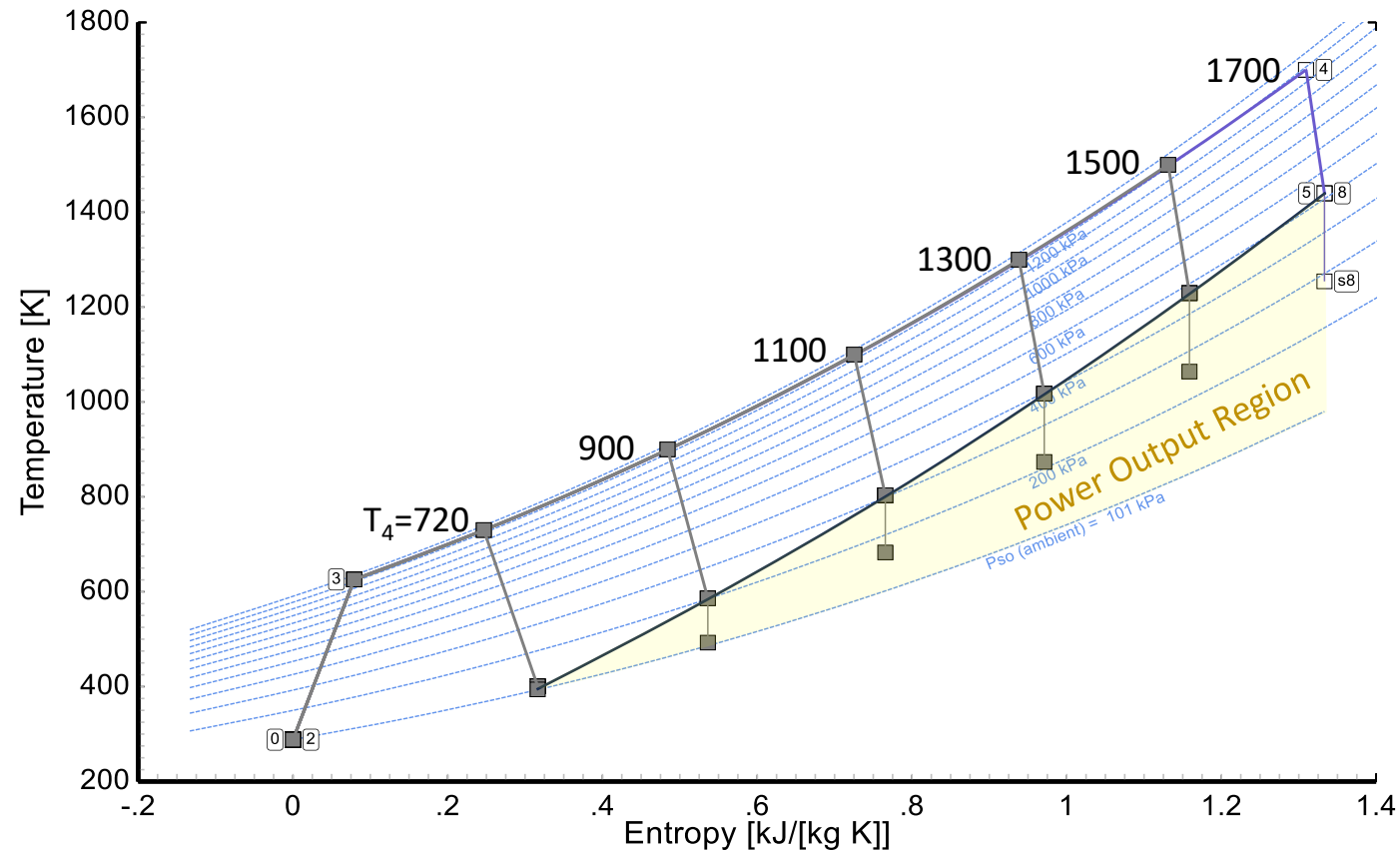
## Specific Power





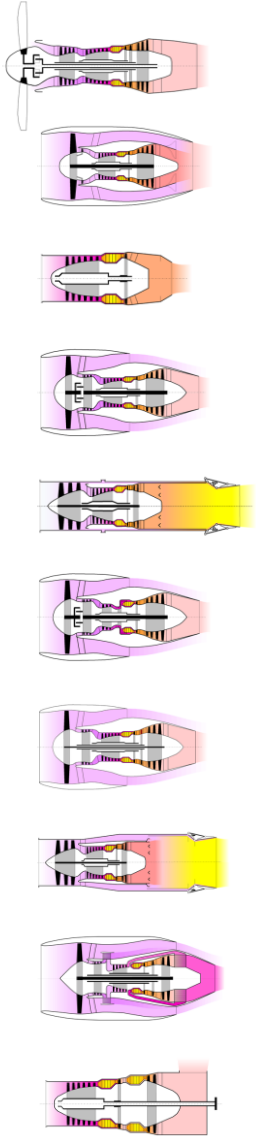
# Why Increases Specific Power with $T_4$ ?

## Pressure Ratio = 12



Because the isobar lines diverge. No equation is needed to explain why specific work increases with burner exit temperature!





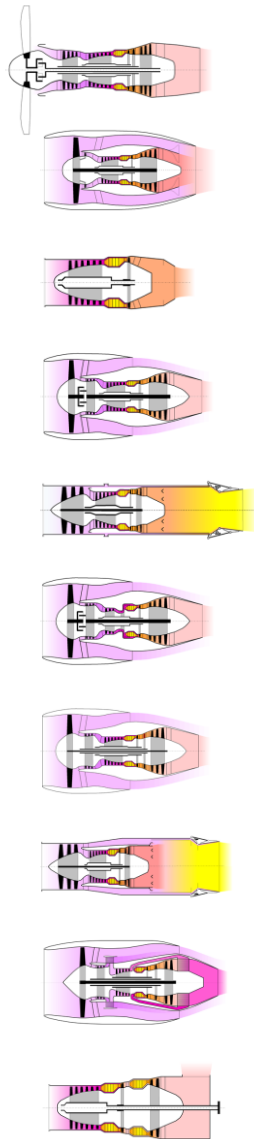
# De-Mystify Your Performance Software

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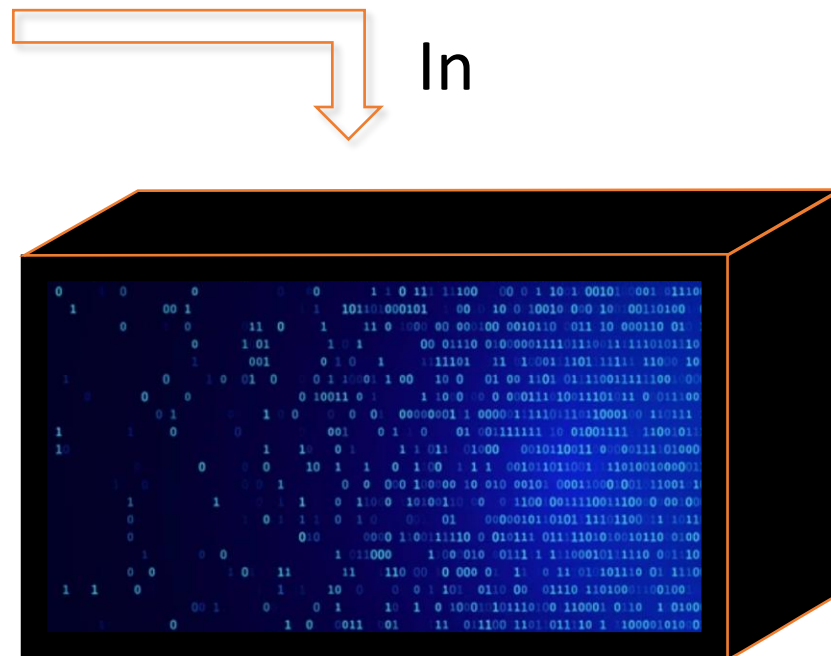
... by doing calculations manually







- Basic Data
- Secondary Air System
- ☒ Ambient Conditions
- ☒ Comp Efficiency
- ☒ Comp Design
- ☒ Turb Efficiency
- ☒ Tip Clear.
- ☒ Reheat
- ☒ Nozzle Selection
- ☒ Nozzle Calculation
- Stations



Out

Station	W kg/s	T K	P kPa	WRstd kg/s
amb				
1	31.680	288.15	101.325	
2	31.680	288.15	101.325	
3	31.680	630.42	1203.741	3.944
31	28.195	630.42	1203.741	
4	28.857	1450.00	1167.629	5.617
41	30.441	1411.20	1167.629	5.846
49	30.441	1113.50	367.374	
5	32.025	1091.37	367.374	17.190
6	32.025	1091.37	360.027	
8	32.025	1091.37	360.027	17.541
Bleed	0.317	630.42	1203.738	
<hr/>				
P2/P1 = 0.9900	P4/P3 = 0.9700	P6/P5 = 0.9800		
Efficiencies: isentr	polytr	RNI	P/P	
Compressor	0.8500	0.8913	0.990	12.000
Burner	0.9999		0.970	
Turbine	0.8900	0.8757	1.798	3.178
<hr/>				
Spoof mech Eff	0.9999	Nom Spd	12499 rpm	

FN	=	26.37 kN
TSFC	=	25.0985 g/(kN*s)
FN/W2	=	832.50 m/s
<hr/>		
Prop Eff	=	0.0000
eta core	=	0.3884
<hr/>		
WF	=	0.66194 kg/s
s NOx	=	0.28659
XM8	=	1.0000
A8	=	0.0773 m²
P8/Pamb	=	3.5532
WB1d/W2	=	0.010000
Ang8	=	20.00 deg
CD8	=	0.9600
WC1N/W2	=	0.050000
WC1R/W2	=	0.050000
Loading	=	100.00 %
e45 th	=	0.87139
far8	=	0.02111
PWX	=	0.00 kW

Is it Magic?



Net Thrust	
Thrust Specific Fuel Consumption	
Specific Thrust	
<hr/>	
Propulsion Efficiency	
Core Efficiency	
<hr/>	
Fuel Flow	
NOx Severity Parameter	
Nozzle Throat Mach No.	
Geometric Nozzle Throat Area	
Nozzle Pressure Ratio	
Bleed Air Flow/Mass Flow W2	
Nozzle Petal Angle	
Nozzle Discharge Coefficient	
Turbine Nozzle Guide Vane Cooling Air / W2	
Turbine Rotor Cooling Air/ W2	
Burner Loading in % of the Cycle Design Point Value	
Thermodynamic Turbine Efficiency	
Nozzle Throat Fuel-Air-Ratio	
Power Offtake	



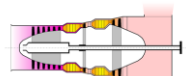
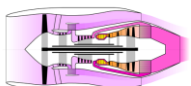
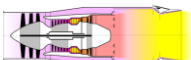
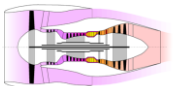
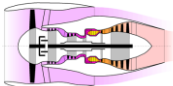
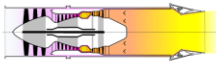
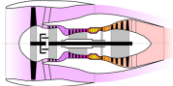
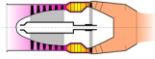
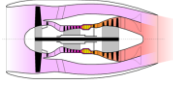
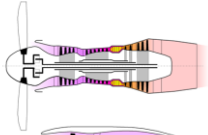
# About Performance Software

- Gas turbine performance calculation programs differ from those of computational fluid dynamics (CFD) and finite element analysis (FEA) in one important aspect:

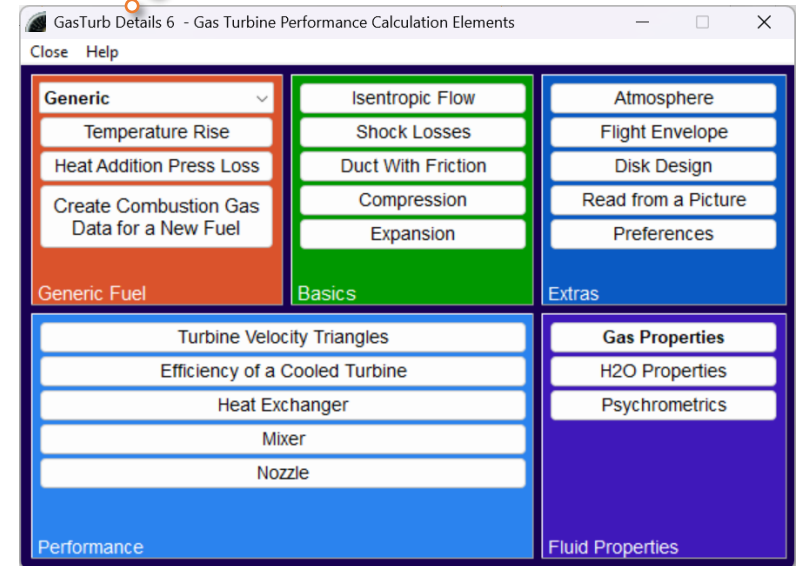
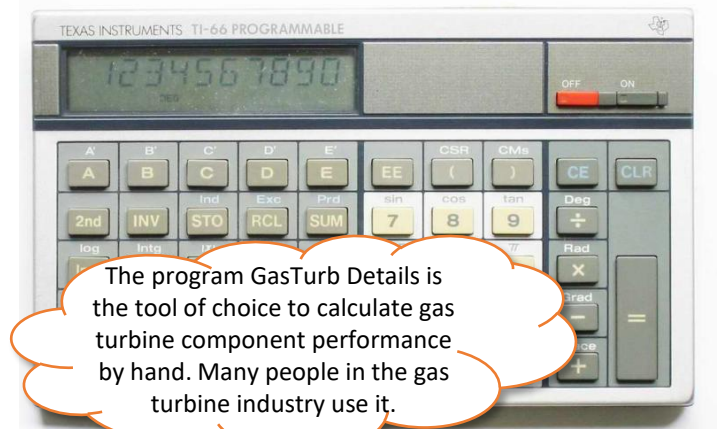
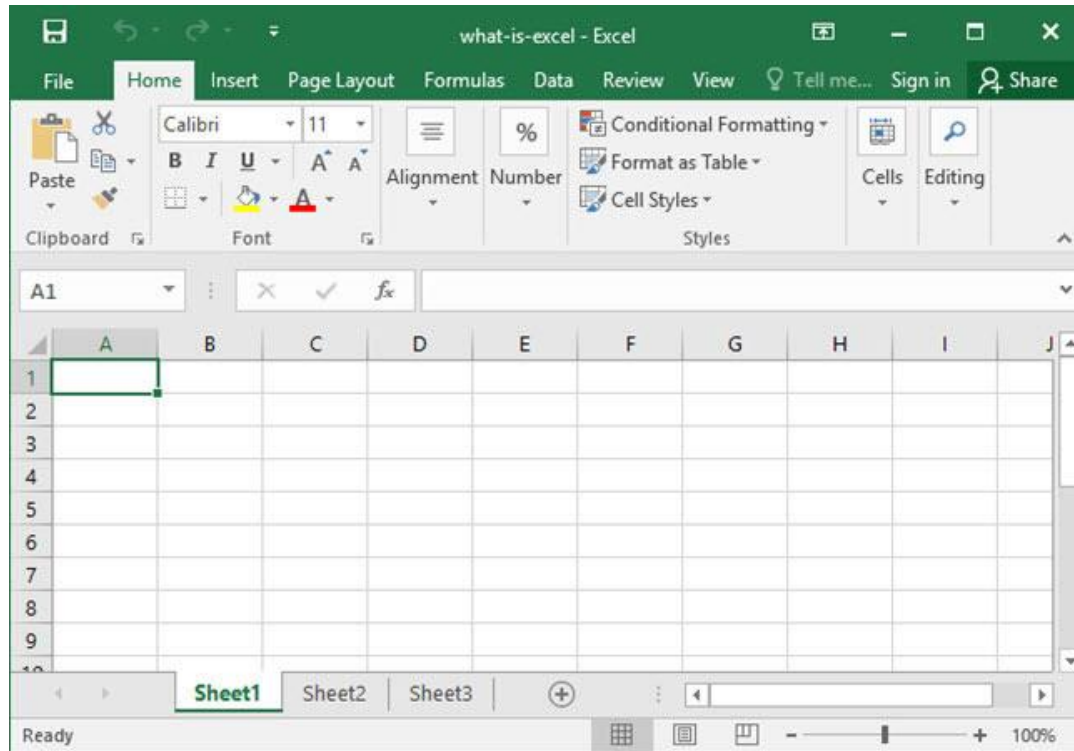
Their output can be easily reproduced on a pocket computer or with Excel.

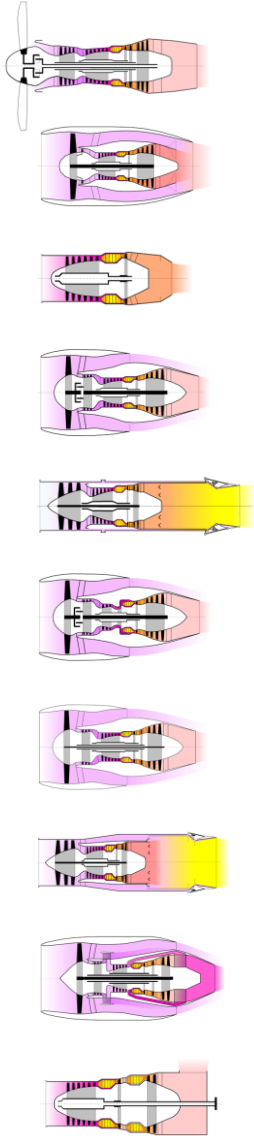
- Doing that is not just an academic exercise, but often necessary in practical engineering when partners in a collaborative project arrive at different results.





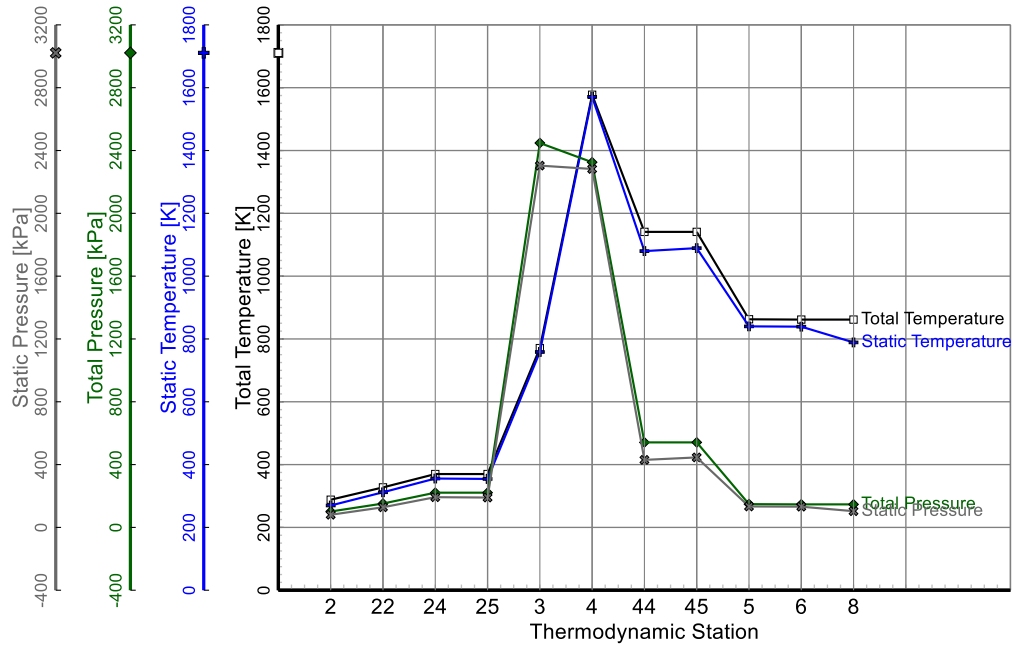
# Calculation of the Cycle by Hand



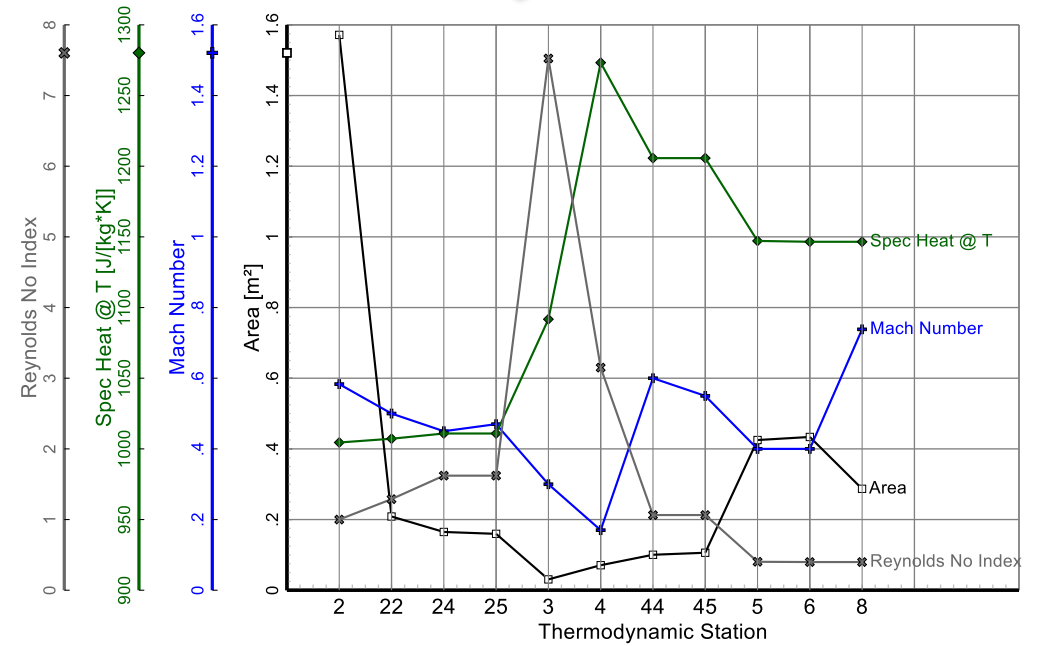


# Cycle Data CFM56-3 Take Off

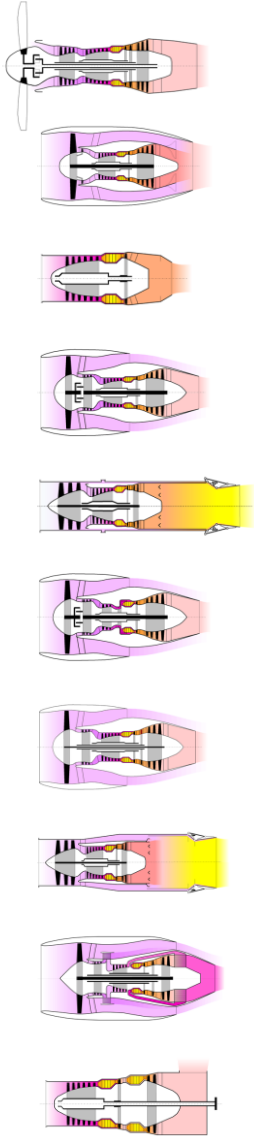
The cycle calculation yields the total pressures and temperatures.



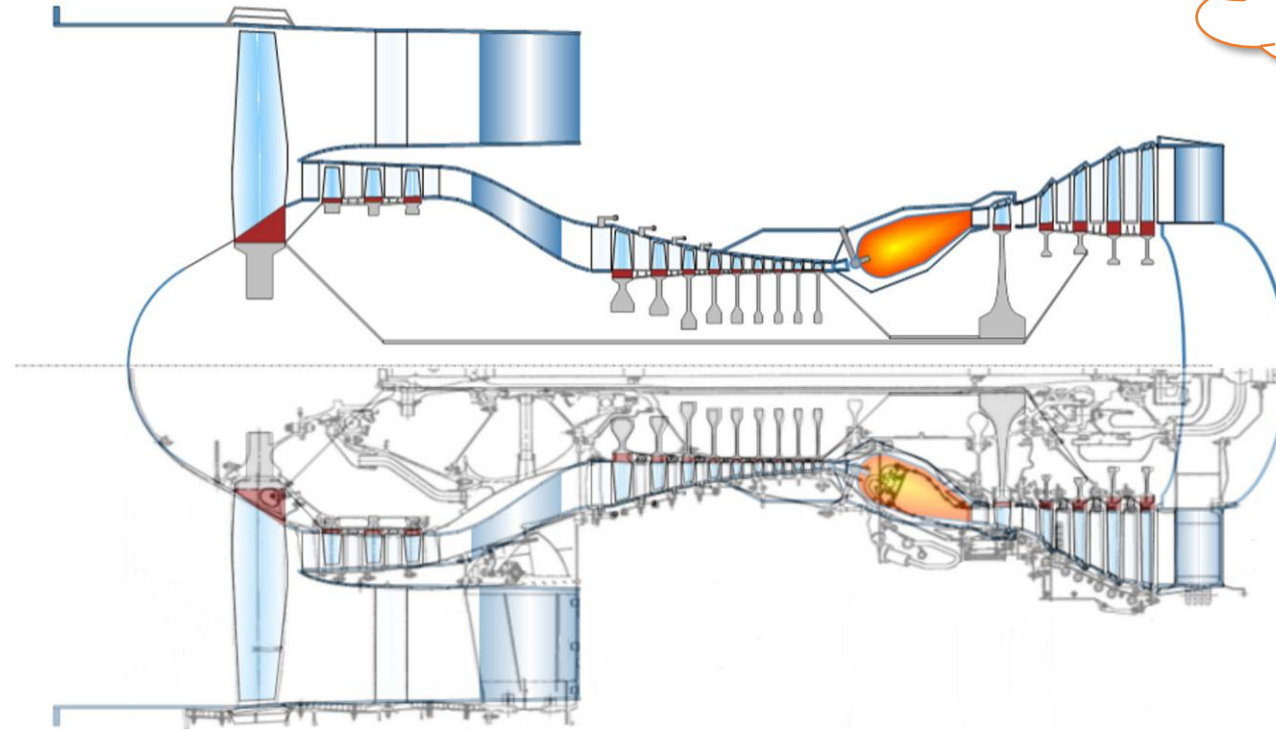
The engine geometry depends on the local Mach numbers.





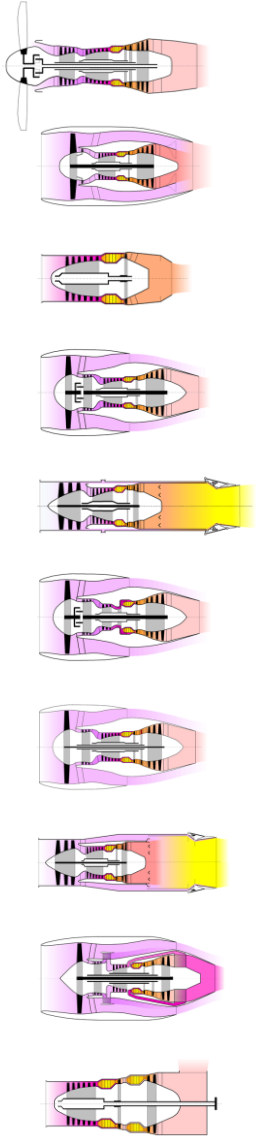


# Geometry Model of the CFM56-3 Turbofan



Students will find it challenging and interesting to reproduce the geometry of an engine based on a given cross-cut and publicly available cycle data.



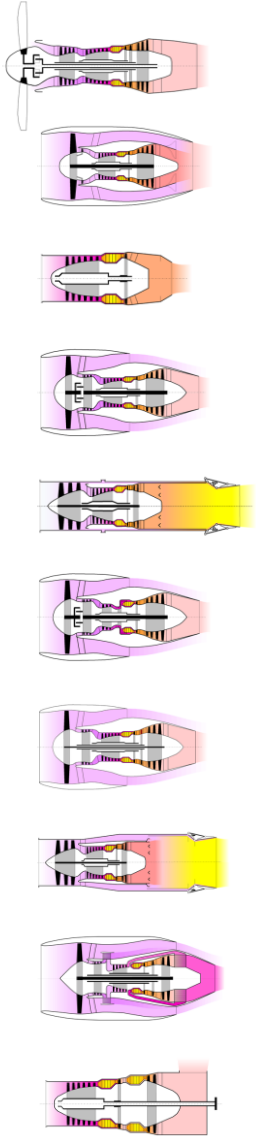


# Visualizing the Results

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... a picture is worth a thousand words





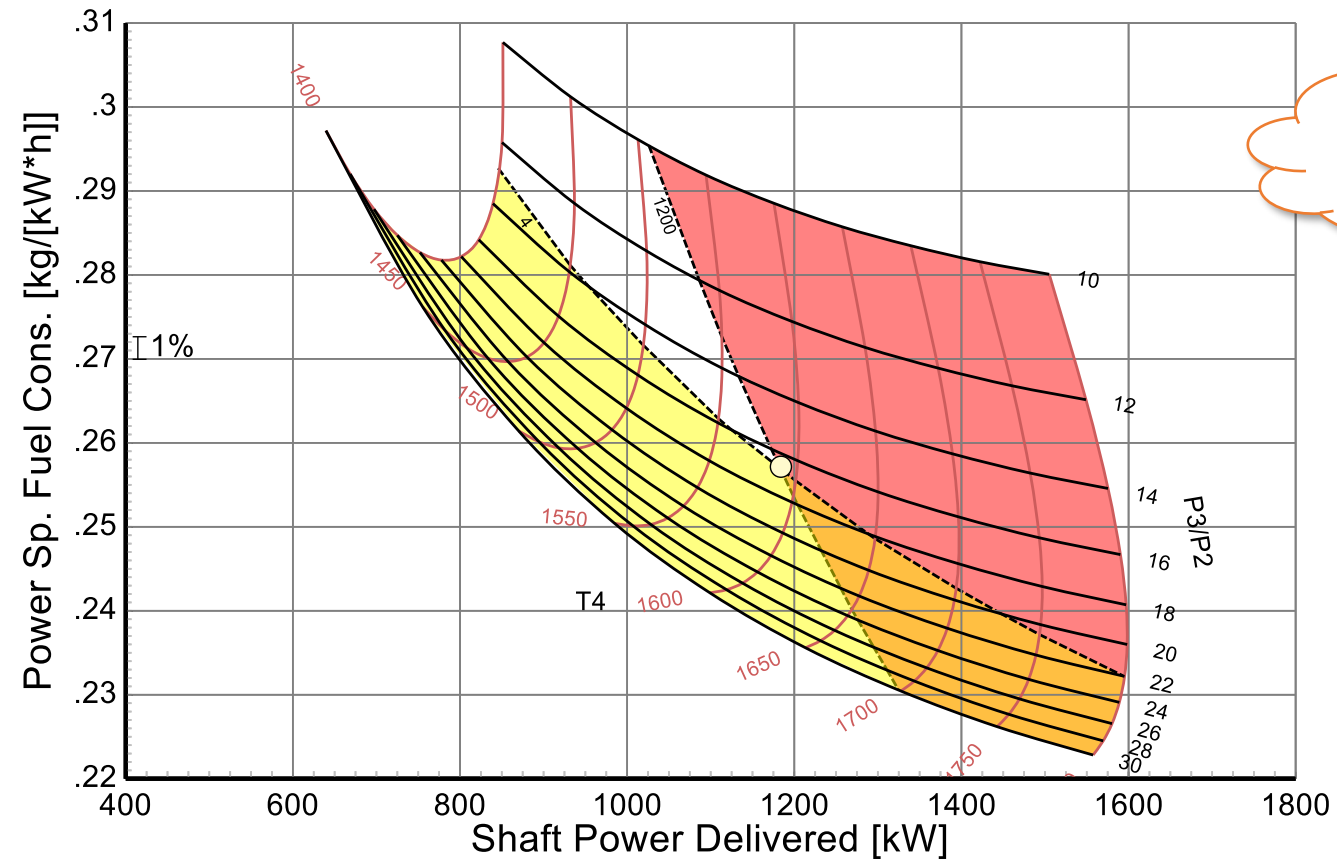
# Parametric Study

## Design Space

Pressure Ratio = 10 ... 30

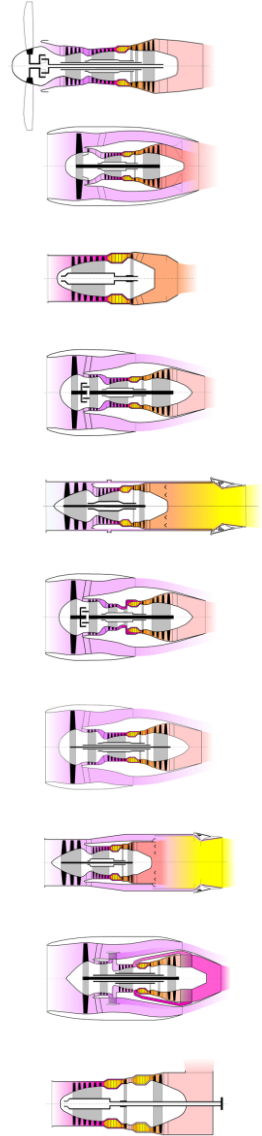
Burner Exit Temperature = 1400 ... 1800 [K]

HP Turbine Pressure Ratio > 4  
HP Turbine Exit Temp T44 > 1200[K]

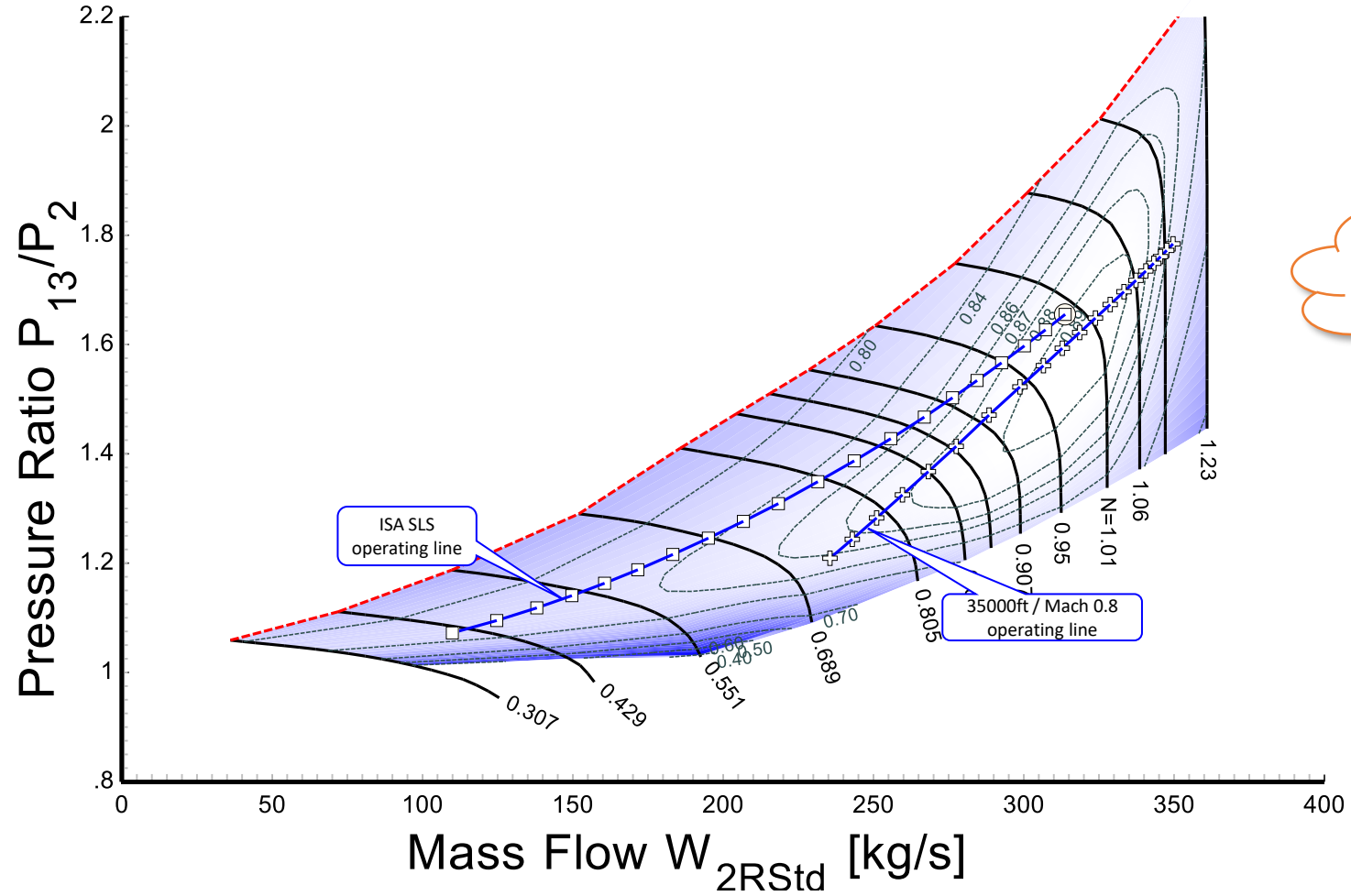


This is a conventional parametric study from which students can learn a lot.





# Operating Lines in the Fan Map



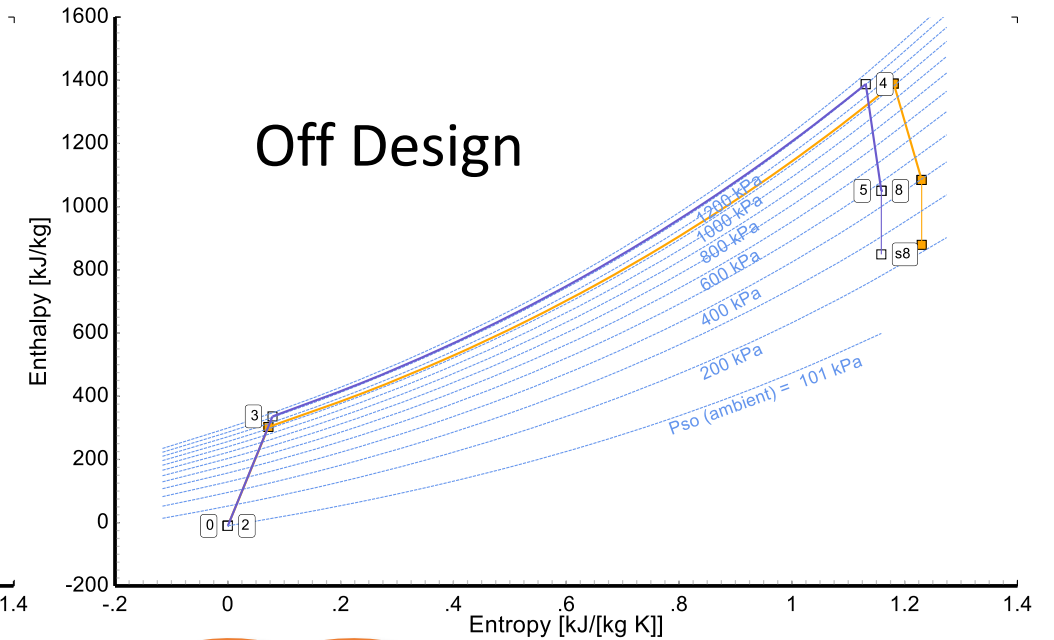
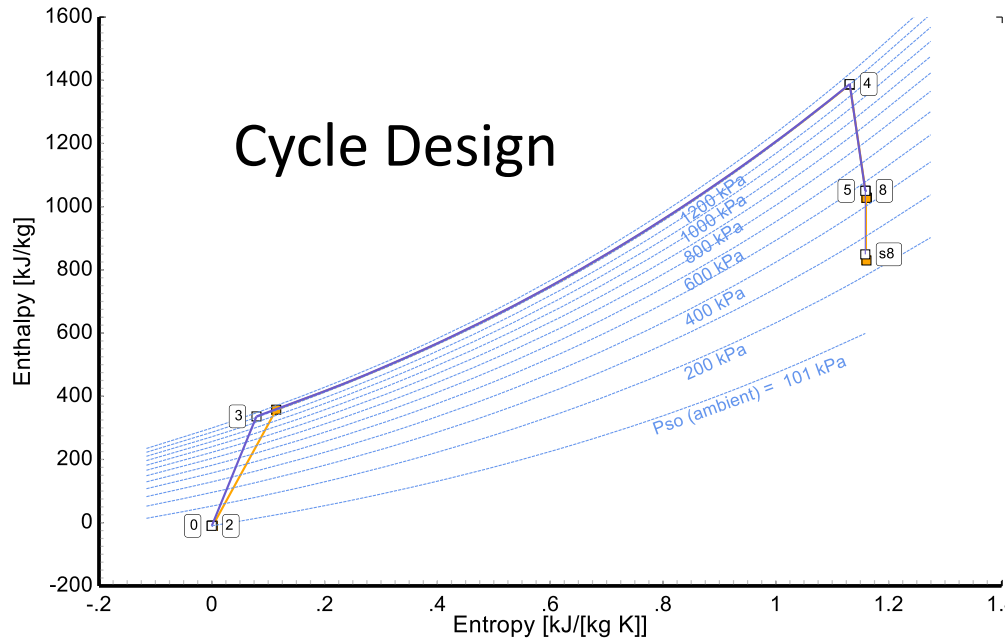
Using colors makes graphics easier to understand and remember.





# Enthalpy-Entropy Diagram

## Effect of -5% Compressor Efficiency



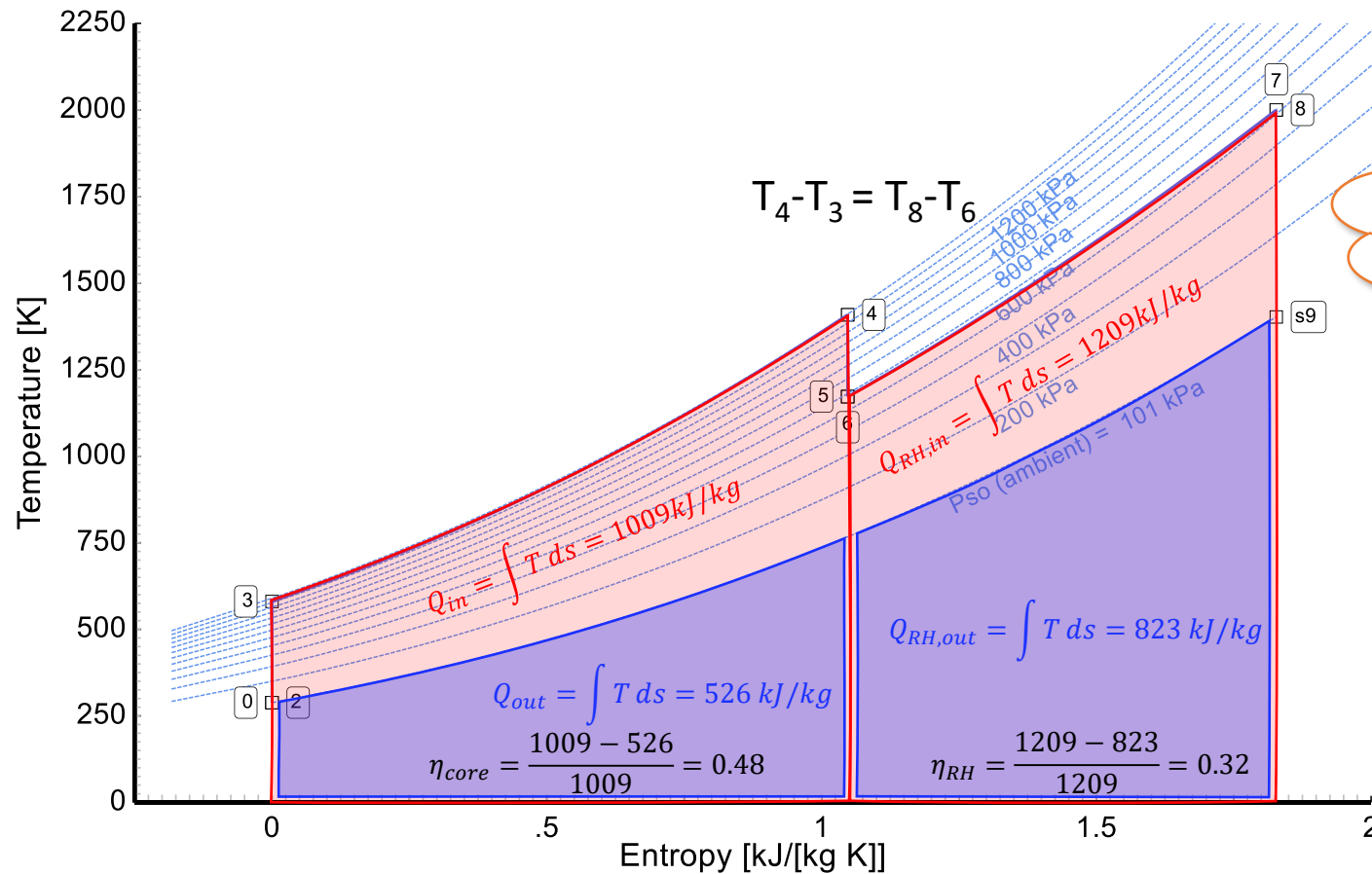
Drawing enthalpy-entropy diagrams to scale can be very helpful for understanding the effects of changes. It is much better than using complex equations!



# Temperature-Entropy Diagram

## Visualize the Amount of Heat Transferred

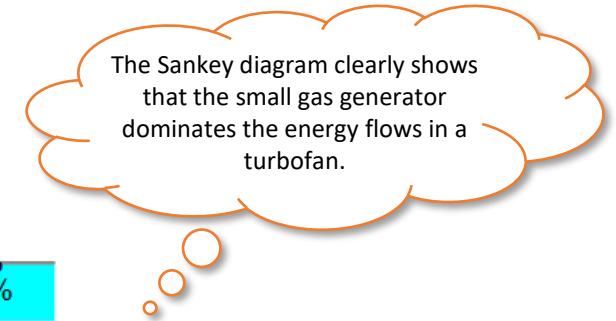
Why is afterburner SFC much larger than dry SFC ?



Drawing temperature-entropy diagrams to scale can help to improve understanding of thermodynamics. Presenting results in graphical form is much more effective than using complex equations!

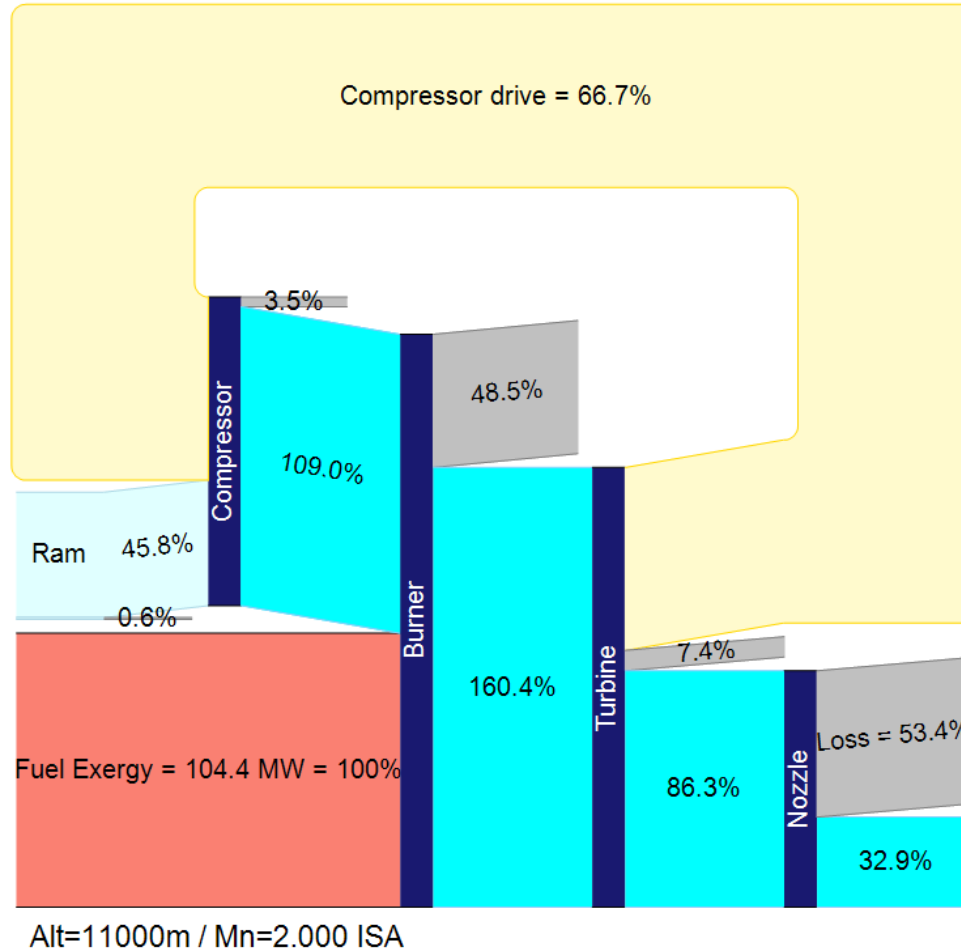


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# Work Potential - Exergy Diagram

## Turbojet, Mach 2, 11km



A turbojet could be an attractive option for the propulsion system of a supersonic fighter aircraft.

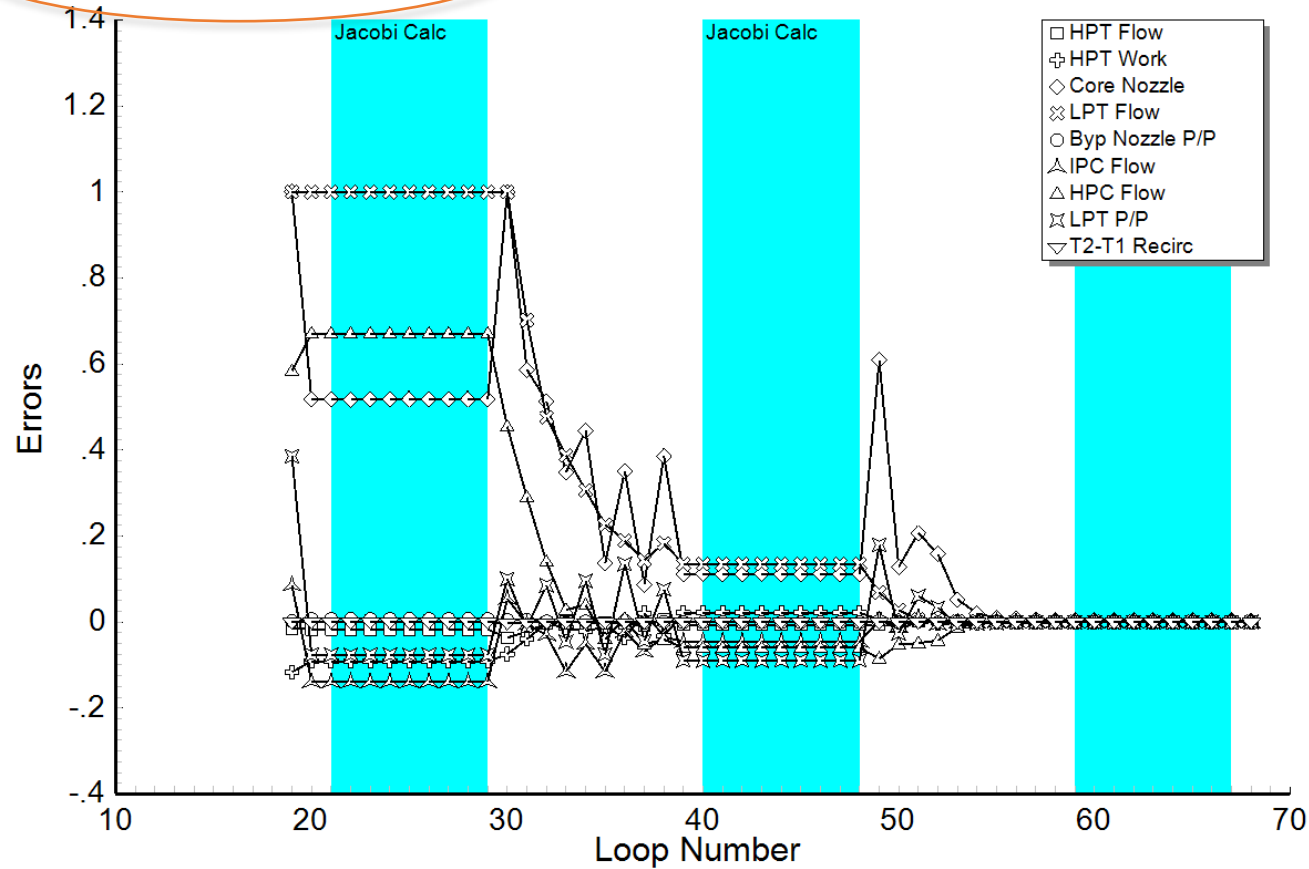
An exergy diagram shows where losses in terms of work potential occur, which is important to consider when analysing the system.





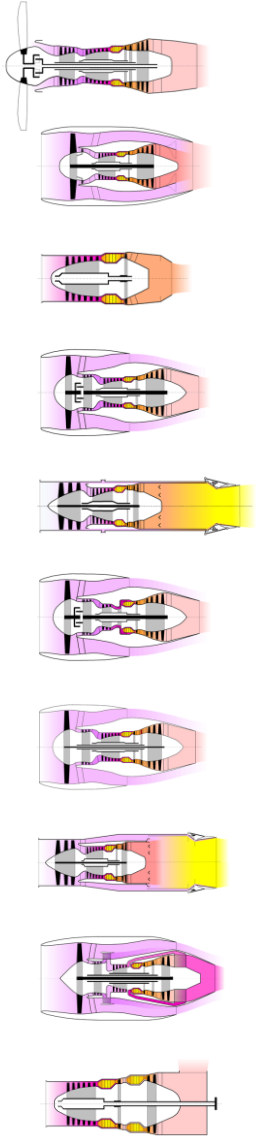
# The Iteration Progress

FROM SL static, ISA , Rel GG Speed=0.580  
TO SL static, ISA , Rel GG Speed=1.000



Off-design calculations require multidimensional, iterative algorithms. Sometimes, convergence problems are encountered. Solving these problems is very important in practice. Graphics can be helpful.





# Is It Magic?

De-Mystify your Performance Program !

That's what a lecturer's really for – not teaching complex equations that don't actually apply to the day-to-day work of a performance engineer.



# A good bedtime read ...

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